

IMPROVED MASSAGING DEVICE FOR CHAIRS

5 CROSS-REFERENCE TO RELATED APPLICATION

This application claims the benefit of U.S. Provisional Application No. 60/148, 929, filed August 5, 1999, the disclosure of which is hereby incorporated by reference.

FIELD OF THE INVENTION

10 The present invention relates to massaging devices, and more particularly, to massaging devices utilizing a greaseless rail system, and/or non-rotary massaging members.

BACKGROUND OF THE INVENTION

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20 Certain custom-built massaging chairs known in the art include a massaging device for performing massaging functions. One type of massaging device is shown in PCT International Application No. PCT/JP99/01340, filed March 17, 1999, by Shimizu Nobuzo. The massaging device used in such chairs includes a track, a massage wheel driving mechanism slidably coupled to the track, and a pair of rotating massage wheels, which are attached to the drive mechanism and translated along the track. The track forms two C-shaped rails. One or more guide wheels having a generally flat circumferential surface are coupled to each side of the driving mechanism. The wheels on each side of the mechanism are fitted within a corresponding rail. Grease is typically applied within the rails to reduce friction between the wheel sides and the rails. The driving mechanism is electrically coupled via electrical wires to a controller that provides the appropriate signal to a motor for driving the mechanism back and forth along the rails. The controller is coupled to a selection device for allowing the user of the massaging chair to turn the motor on and off and to select the speed of the movement of the massaging wheels. The driving mechanism generally includes a limit switch, which controls the motion of the driving mechanism along the rails.

25 Each massaging wheel is coupled to the driving mechanism about a rotary shaft. The massaging wheels are mounted to the rotary shaft eccentrically, and in an oblique fashion relative to the spin axis of the shaft. A second motor rotates the massaging wheels. The wheels are mounted eccentrically and obliquely relative to the spin axis, allowing the outer-peripherals of the massaging wheels to move from side-to-side in a reciprocating fashion. As the driving mechanism travels along the rails, it enables the massaging wheels to translate longitudinally, while the motor causes the wheels to simultaneously move back and forth sideways.

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The massaging device is typically located in the back of the chair, with the rails running vertically along the back of the chair and with the massaging wheels making contact with the fabric on the front face of the chair. Thus, the user sitting in the chair comes in indirect contact with the massaging wheels. Typically, the massaging device is centered along the back of the chair so as to straddle the spine of the user. As the driving mechanism rides up and down along the rails, the massaging wheels massage the user's back as they move longitudinally and sideways along the back of the chair.

A problem with existing massaging devices is that with time, wear of the guide wheels causes the guide wheels to rattle within the rails during operation, which may result in an annoying clattering sound. In addition, current massaging devices are often wearing on the chair fabric. As the massaging wheels translate longitudinally along the length of the chair, the wheels' sideways motion exerts lateral frictional forces on the fibers of the chair's fabric, causing the fibers to tear over time. In a similar fashion, wheel rotation exerts longitudinal forces on the fabric, which also tends to abrade or tear the fabric over a period of time.

Current massaging devices are also hazardous. As the rotating wheels move from side-to-side, the outer-periphery of the wheels rotate in close proximity to the drive motor, creating a pocket whereby objects may be crimped. Because of the compliant characteristics of the chair fabric that is interposed between the user and the massage wheels, the user's limbs or parts of their flesh may be pinched within the pocket, creating a potential hazard.

Existing massaging devices also do not adequately protect the wiring that sends signals and provides the power to drive the driving mechanism from becoming tangled and chaffed from the movement of the driving mechanism. Tangled and chaffed wires may result in failure of the massaging device and sometimes in hazardous conditions such as the initiation of a fire. Moreover, the driving mechanism limit switches in these devices are openly exposed, leading to the risk of damage or misalignment, either of which may result in subsequent malfunction or damage to the massage mechanism.

Another problem inherent in conventional massaging devices that use grease to induce smooth travel of the guide wheels within the rails, is that the grease can escape the rails and stain the chair. Grease also accumulates dirt and dust, which deteriorates the performance of the massaging device over time. Additionally, current massaging devices are bulky in size and weight. The bulky profile of current massage devices require massage chairs using these devices to grow in size and weight, making it difficult to incorporate the device into chairs having small profiles, such as the bucket seats of cars and aircraft.

Moreover, current messaging devices incorporated within reclining chairs are not modular. When the messaging device requires maintenance, either a technician is required to service the reclining unit at the customer's residence, or the reclining chair, as a unit, must be transported to the service center. Thus, servicing current messaging units can be costly and inconvenient.

What is needed, therefore is a massaging device that preferably does not rattle with age, does not wear away the chair fabric at a considerable rate, and is safe to the user. Such a device preferably provides protection to the wiring between the driving mechanism and the controller against chaffing, provides protection to the driving mechanism limit switches to prevent switch damage or misalignment, and is more compact than current massaging devices. Further, such device is modular, providing convenient and inexpensive maintenance.

15 SUMMARY OF THE INVENTION

The present invention provides, in one embodiment, a massaging device having a track comprising two rails. The device also includes a driving mechanism that causes a massaging unit comprising a pair of massaging members to move back and forth along the rails. Each rail is positioned at an obtuse angle relative to the plane of the driving mechanism, creating a first V-shaped raceway when viewed from an end of the massaging device. A second raceway on an inner surface of the rail is preferably parallel and spaced apart from the plane of the driving mechanism. The driving mechanism may include a carriage in which two guide wheels extend from each side of the carriage. Each guide wheel is tapered, having a generally diamond shaped cross-section such that each wheel may be mated to travel along the first raceway of each rail. A biasing wheel pivotally coupled on either side of the carriage, is positioned between and spaced apart from the two guide wheels on either side of the carriage. The biasing wheel is spring loaded in a direction away from the guide wheels. The carriage slidably fits within the track such that the guide wheels fit within the corresponding first raceway, while the biasing wheel is spring loaded into a position bearing against the second raceway of its corresponding rail. The biasing wheel insures that the carriage is maintained within the rails, thereby taking up any slack that would otherwise form due to wear of the guide and biasing wheels. Moreover, with the use of tapered guide wheels, a smooth movement of the guide wheels within the track is obtained, alleviating the need to grease the rails.

A threaded guide rod, rotably attached to a drive motor, is incorporated in the track and spans the length of the track. The guide rod engages a cylindrical member coupled to the driving

mechanism so as to translate the driving mechanism along the rod as the rod is rotated. A controller, which receives signals from a user control or remote control, controls the translation of the driving mechanism and massaging device.

In a another embodiment, a massaging unit is coupled to the driving mechanism. The massaging unit comprises a pair of left and right massaging members mounted on an intermediate portion of a rotary shaft in a canted fashion relative to the an axis of the rotary shaft, and a half-turn clutch for selectively switching the motion of the pair of right and left massaging members between a kneading to non-kneading motion. In the kneading motion, where the pair of massaging members are slanted opposite to each other, the massaging members move towards and away from each other as the rotary shaft rotates in a first direction. In the non-kneading motion, the massaging members move in parallel as the rotary shaft rotates in a direction opposite the first. The massaging members are partial discoid in shape having a lobe which extends from a central portion of the member. The massaging unit may include a retaining apparatus for limiting the rotation of the massaging members relative to the rotary shaft. Additionally, the lobes may be configured into the shape of a finger or fist. Further, the lobes may be either fixed or detachable elements.

The massaging device according to the present invention is modular and may be incorporated in various types of massaging apparatus' such as a massaging chair, or a stand-alone one piece casing that may be leaned against a wall or the back of a chair. In further embodiments, the massaging device is hand-carriable, wherein the massaging unit is housed within a simple casing instead of traveling along a track.

The present invention may readily retrofit existing recliners. The invention's improved size and weight provides advantages over massaging devices of the prior art. The present invention's greaseless operation and durable construction provides additional advantages over the prior art. Further, the massaging members of the present invention are configured such that they do not rotate in close proximity to the structure of the massaging unit. Accordingly, fingers or other body parts will not become pinched between the support frame of the massaging unit and the massaging members.

DESCRIPTION OF THE DRAWINGS

These and other features and advantages of the present invention will be better understood by reference to the following detailed description when considered in conjunction with the accompanying drawings wherein:

FIG. 1 is a front view of a massaging device of the present invention;

FIG. 2 is a enlarged partial front view of the massaging device shown in FIG. 1;

FIG. 3 is a side view of the driving mechanism of the massaging device of the present invention;

FIG. 4 is a bottom view of a side end of the driving mechanism shown in FIG. 3;

FIG. 5 is a top view of a massaging device of the present invention;

FIG. 6 is a partial top view of the driving mechanism installed on the massaging device of the present invention;

FIG. 7 is an exploded view of a massaging member assembly incorporated in the carriage shown in FIG. 6;

FIGS. 8A and 8B are front and side views of an embodiment of the massaging member according to the present invention;

FIGS. 9A and 9B are enlarged partial perspective views of preferred and alternate embodiments of the retaining apparatus incorporated in the embodiments shown in FIG. 8;

FIG. 10 is an end view of an alternate embodiment of a massaging device carriage assembly according to the present invention;

FIG. 11 is an enlarged partial cross sectional view of the clutch mechanism incorporated in the embodiment shown in FIG 13.

FIG. 12 is a cross sectional view of the section of the clutch shown in FIG. 17 taken along line 16-16;

FIGS. 13A and 13B illustrate the massaging members in parallel, non-kneading motion;

FIGS. 14A and 14B depict the massaging members of the present invention in nonparallel, kneading motion;

FIG. 15 is a perspective view of a conventional recliner incorporating the massaging device of the present invention;

FIG. 16 is an exploded perspective view of the adjustable fastener used to secure the massaging device to the recliner shown in FIG. 15.

FIG. 17 is a perspective view of the recliner incorporating the massaging device shown in FIG. 15;

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FIG. 18 is a partial side view of the back of the recliner shown in FIG. 15.

FIG. 19 is a partial perspective view of the back of the recliner shown in FIG. 15.

FIG. 20 is a schematic view of a massaging device incorporated in a stand alone unit leaning against a wall;

FIG. 21 is a schematic view of a massaging device incorporated in a stand alone unit and leaning against the back of a chair; and

FIG. 22 is a partial end view of a massaging device incorporating additional multiple smaller massaging wheels.

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DETAILED DESCRIPTION OF THE INVENTION

Referring to FIG. 1, in a preferred embodiment of the invention, a massaging device 7 includes a track comprising two rails. The massaging device 7 also includes a driving mechanism 5 that causes a massaging unit 6 comprising a pair of massaging members 116R, 116L to move back and forth along the rails. Preferably, the rails are part of a unitary track structure 30 comprising a bracket 32 having proximal and distal ends 250 and 260 (see FIG. 5), and rails 34 formed on opposite sides of the bracket 32. Because both rails are preferably identical, only one of the rails is described herein for convenience.

Referring to FIG. 2, the rail 34 comprises a channel shaped cross-section and is positioned at an acute angle 36 relative to the plane 33 of movement of the driving mechanism 5. The rail 34 has a first leg 38 spanning the length of the rail 34. From the first leg 38 extends a web 40 that spans the length of the rail 34. The web 40 is preferably perpendicular to the first leg 38. A second leg 42 extends perpendicularly from the web 40 opposite the first leg 38 whereby the first leg 38, the web 40 and the second leg 42 define a channel 44. A first lip portion 46 extends from the second leg 42 at an obtuse angle 48 towards the first leg 38. The first lip portion 46 spans the length of the rail 34. A second lip portion 50 extends from the first lip portion, spanning the length of the rail 34. The second lip 50 preferably extends at an angle such that it is perpendicular to the plane of movement 33 of the driving mechanism 5.

A first raceway 52 is defined in the rail 34 between the first leg 38 and the web 40. Because of the angle 36 of extension of the rail 34 relative to the plane 33 of movement of the driving mechanism 5, the first raceway 52 is V-shaped in cross-section when viewed from an end of the massaging device 7. A second raceway 54 is defined on the inner surface of the first lip portion 46. Each rail 34 is preferably formed from a single sheet of material, for example, by bending a single sheet of metal. In the preferred embodiment shown in FIGS. 1 and 2, the entire track 30 is formed from a single sheet of metal. In alternative embodiments, the track 30, may comprise injection molded polished plastics such as delrin, Teflon and the like. In other embodiments, the track 30 may comprise ceramic materials having polished surfaces and high tensile strengths. In another embodiment, the two rails 34 can be separate structures that are interconnected defining a track 30.

Referring to FIG. 3, the driving mechanism 5 comprises a carriage 56. The carriage 56 supports an axle 58 onto which are mounted the massaging members 116L, 116R. Preferably, a set of guide wheels 60 extend from each side of the carriage 56. Corresponding wheels 60 on each side of the carriage 56 may be coupled to the same axle. For example, in the preferred

embodiment, one set of wheels 60 is coupled to a first axle 62 and another set of wheels 60 is coupled to a second axle 64. In an alternate embodiment, a separate axle may be provided for each wheel 60.

Referring to FIG. 7, each guide wheel 60 has a sidewall surface 66 which tapers inward such that each guide wheel 60 has a generally diamond shaped cross-section. An annular groove 65 formed along a vertex 67 of each guide wheel 60 accommodates an O-ring 68 preferably made from rubber or other similar material.

As illustrated in FIG. 2, the tapering of the sidewalls 66 is such that each wheel 60 can be mated to the first raceway 52 of each rail 34. As such, the rubber or rubber-like O-ring 68 rides at the vertex 70 of the first raceway 52. Each guide wheel 60 is preferably double molded with its interior molded from nylon, and its exterior (or overmold) molded from urethane. The nylon center acts as the bearing bushing that fits over a guide wheel axle 62,64 (FIG. 6) whereas the softer urethane outer surface serves to increase grip and significantly reduce vibrations and noise as the wheels 60 travel along the rails 34.

Referring now to FIGS. 3 and 4, a biasing wheel 72 is coupled on either side of the carriage 56. Preferably each biasing wheel 72 is positioned between the two guide wheels 60 on either side of the carriage 56. Preferably, each biasing wheel 72 comprises a first larger diameter section 74 and second smaller diameter section 76. The second smaller diameter section 74 extends axially and concentrically from the first section 74. Because of its function, the biasing wheel 72 preferably comprises a bearing material, such as Nylon, Delrin, Teflon or other materials having similar mechanical properties. In preferred embodiments, the second section 74 is overmolded with rubber or a rubber-like material 78, such as urethane. In alternative embodiments, a rubber or rubber like O-ring is fitted within an annular groove formed along the circumferential surface of the second section 74 of each biasing wheel 76.

Each biasing wheel 72 is mounted on an axle 80 which is perpendicularly mounted on a pivoting arm 82. The pivoting arm 82 is pivotally coupled to a side of the carriage 56 via an axle 84, and is spring loaded in a direction away from the guide wheels 60. This may be accomplished using a torsion spring assembly 85 coupled to the pivoting arm 82 and carriage 56 in surrounding relationship with the axle 84. Alternatively, an axial spring (not shown) may be used that is coupled to the carriage 56 and transversely to the pivoting arm 82 for biasing the pivoting arm 82 in a direction away from the guide wheels 60. Other spring mechanisms are known in the art and may also be used.

As described in FIG. 2, the carriage 56, with massaging unit 6, is slidably coupled within the track 30 such that the guide wheels 60 are fitted within the corresponding first raceway 52 of each rail 34 while the second section 76 of each biasing wheel 72 is biased by the spring loaded arm 82 into a position bearing against the second raceway 54 of its corresponding rail 34. The first section 74 of each biasing wheel 72 bears against the inner surface 86 of second lip 50 of its corresponding rail 34, providing secondary alignment of the carriage 56 along the rail 34. The biasing wheels 72 are biased in a direction opposite the location of the guide wheels 60 to insure that the carriage 56 is maintained within the rails 34. By being spring loaded, the biasing arm 82 always biases the biasing wheel 72 against the second raceway 54, thereby taking up any slack that would otherwise form due to wear of the guide and biasing wheels. Consequently, the biasing wheels 72 are self-adjusting, taking up all the slack caused by wheel wear and alleviating the rattling that results from such slack. In addition, the use of the rubber or rubber-like O-rings on the guide wheels 60 serves to reduce vibration and noises as the carriage 56 rides along the track 30. This type of vibration is further reduced by the use of a softer material such as urethane to form the outer surfaces of the guide wheels 60 as described above. Moreover, the tapered guide wheels 60, i.e., guide wheels that have a generally diamond shaped cross-section, riding in a V-shaped raceway provide sideways containment of the massage carriage 56 without the need to use lubricants, as opposed to the conventional C-profile, which need lubrication due to the requirement for tight tolerances.

In alternative embodiments, the carriage 56 may be outfitted with more than one biasing wheel 72 on either side. Moreover, one or more guide wheels 60 may be used on either side of the carriage 56. Furthermore, each biasing wheel 72 may only comprise a section that rides on the second raceway 54 of a rail 34. In such case, a second lip 50 need not be formed on the rails 34.

Referring to FIG. 5, a guide rod 90 is preferably incorporated in the track 30, spanning the length of the track 30. A cylindrical member 92 coupled to the driving mechanism 5 fits over the rod 90 such that the rod 90 penetrates the cylindrical member 92. In this regard, the rod 90 also serves to guide the driving mechanism 5 along the track 30. The guide rod 90 has a threaded outer surface while the cylindrical member 92 has a threaded inner surface mating with the outer surface of the guide rod 90. In one embodiment, the guide rod 90 is rotably attached to a drive motor (not shown), which causes the guide rod 90 to rotate and thread through the cylindrical member 92 so as to move the driving mechanism 5 along the rod 90. By reversing the rotation of the guide rod 90, the driving mechanism's 5 path is reversed. In another embodiment, a motor

attached to the driving mechanism 5 causes the cylindrical member 92 to rotate, threading the rod 90 so as to move the driving mechanism 5 along the rod. In a further embodiment, the driving mechanism 5 can drive the guide wheels for translation along the track 30.

To prevent damage to wires providing signals and power to the driving mechanism 5, a flexible conduit 94 is used for harnessing and protecting the wires. To protect the conduit from wearing against the rail edge during movement of the driving mechanism 5, a plastic or rubber-like cover 96 (FIG. 1) is placed over the edge of the second lip 50 of the rail 34 over which the conduit 94 is routed. The cover 96 spans a portion of the second lip 50 length proximate the location of the conduit 94. In embodiments utilizing biasing wheels 72, wherein the first section 74 of the biasing wheel 72 bears against the inner surface 86 of the second lip 50, the cover 96 height is preferably limited to prevent interference with the travel of the biasing wheel 72. In alternative embodiments, clips 98 may be formed or attached on the rail 34 for retaining the conduit 94 close to the rail 34.

To protect the limit switches of the massaging device 7, the present invention incorporates a cover 99 to protect them from damage and misalignment.

The present invention also includes a controller 102 that is coupled to the driving mechanism 5. The controller 102 receives signals from a user control or a remote control 104 for controlling the operation of the massaging device 12.

Referring to FIG. 6, the massaging unit 6 comprises right and left massaging members 116R, 116L. The right and left massaging members 116R, 116L include respective boss portions 115 which are mounted on and rotate with a shaft 110. The massaging members 116R, 116L are rotably coupled to the boss portions 115 along on oblique axis 117, where the boss portion can rotate relative the massaging members 116R, 116L and wherein the massaging members 116R, 116L are eccentrically coupled to the shaft 110. The massaging members 116R, 116L are held by the corresponding boss portions 115 as slanted relative to the axis of the rotary shaft 110.

As shown in FIG. 7, each of the boss portions 115 includes a pair of sandwiching plates 115a and 115b, each in the form of a section of a cylinder sectioned askew relative to the axis of the cylinder, and a central plate 115c interposed between the sandwiching plates 115a and 115b. The central plate 115c is a shaped discoid with its opposite sides respectively abutting the slanted end faces of the sandwiching plates 115a and 115b. The plates 115a, 115b and 115c attach to the massaging member 116 by placing the sandwiching plates 115a and 115b on opposite sides of the massaging member 116, while the massaging member 116 centrally receives the central plate 115c. The plates 115a, 115b and 115c are fastened to the messaging member with bolts 118 which

extend through the three plates and nuts 119. In one embodiment, the central plate 115c is formed integrally with one of the sandwiching plates 115a and 115b. Alternatively, the central plate 115c may comprise mating halves, having half the thickness of the plate 115c, formed integrally with the sandwiching plates 115a and 115b, respectively.

As depicted in FIGS. 8A and 8B, the massaging members 116R, 116L are each partially discoid in shape, comprising a lobe 113 having a substantially radial cross-section and extending from a central portion of the member 116R, 116L. The massaging members 116R, 116L also include a central hole 116a in the central portion thereof for slidably receiving the central plate 115c for rotation relative to the central plate 115c. Thus, the massaging members 116R, 116L are rotatably coupled to the rotary shaft 110 being slanted relative to the axis of the rotary shaft 110. The massaging members 116R, 116L are designed such that the lobes 113 travel in a reciprocal, sideways motion. Thus, the present invention alleviates potential hazards to the user of pinching flesh between rotating massaging members and the structure of the massaging unit. Further, the partial discoid shape of the massaging members 116R, 116L provides the massaging unit 6 with a thinner profile than rotating massaging wheels of the prior art, as only the portion of the massaging member 116 that contacts the affected part of the user requires a larger peripheral.

The massaging members 116R, 116L are preferably made of a polished plastic, such as Delrin, Teflon or the like. The polished plastic composition provides smooth contact between the massaging members 116R, 116L and covering fabric. The smooth contact reduces the friction between the fabric and massaging members 116R, 116L, and thus, reduces wear on the fabric.

As shown in FIG. 9A, a retaining apparatus 145 extending from a support frame 146 of the massaging unit 6 just beneath the massaging member 116R, 116L is used to constrain the motion of massaging members 116R, 116L to a reciprocal, side-to-side motion. The retaining apparatus 145 comprises a U-shaped retaining bar 147 forming a slot 148 in-line with the axis of the rotary shaft 110. A protruding, bar shaped element 149 formed at a base portion of the massaging members 116R, 116L, slidably engages the slot 148, restricting the massaging members 116R, 116L from continuous rotation with the rotary shaft 110, and limiting the movement of the members 116R, 116L to an oscillating sideways motion. Preferably, the protruding element 149 extends from the side of the massaging members 116R, 116L to aid in further reducing the massaging unit 6 profile. In an alternate embodiment, as shown in FIG. 9B, a tension spring, coupled to the support frame 146 and a peg 151 located along a base portion of the massaging members 116R, 116L, may be used to limit the members 116R, 116L from rotating with the rotary shaft 110.

In a preferred embodiment, as shown in FIG. 6, a separate motor 131 drives the rotary shaft 110, and actuates the massaging members 116R, 116L, while the guide rod 90 and internally threaded cylindrical member 92 move the entire mechanism 5 to a different location along the rails 34. Referring to FIG. 10, the rotary shaft 110 includes a first shaft portion 110L supporting the left massaging member 116L, and a second shaft portion 110R supporting the right massaging member 116R. The second shaft portion 110R is coaxially aligned with the first shaft portion 110L. The first shaft portion 110L comprises a portion for mounting the left massaging member 116L and a portion coupled to the drive element 114 of the drive motor 131. The second shaft portion 110R comprises a portion for mounting the right massaging wheel 116R. The rotary shaft 110 is divided into the first and second shaft portions 110L and 110R at a dividing end 110a located between the drive element 114 and the right massaging member 116R. The dividing end portions 110b and 110c of the first and second shaft portions 110L and 110R are preferably interconnected through a half-turn clutch 121.

As shown in FIG. 11, the half-turn clutch 121 includes a tubular member 120 unrotatably and coaxially secured to the dividing end portion 110c of the second shaft portion 110R, and a stopper pin 125 projecting radially outwardly of the dividing portion 110b of the first shaft portion 110L coaxially and rotatably inserted into the tubular member 120. The tubular member 120 is shaped cylindrical having a bore 122 axially extending through a central portion thereof, and a bearing 123 located on a peripheral edge portion of the opening adjacent the drive element 114 for receiving the dividing end portion 110b of the first shaft portion 110L for rotation. Further, the tubular member 120 is formed in an axially intermediate portion thereof with a semicircular transverse slot 124 which has a length circumferentially of the tubular member 120 corresponding to a half turn and which has a depth from the outer peripheral surface of the tubular member 120 to the bore 122. The stopper pin 125 is secured to the dividing end portion 110b of the first shaft portion 110L by, for example, thread engagement of a setscrew so as to project radially outwardly, and the tip portion of the pin 125 movably stays within the transverse slot 124.

The tubular member 120 defines in a right-hand side end portion thereof a tapped hole 128 for thread engagement with a setscrew 127 preventing the dividing end portion 110c of the second shaft portion 110R from rotating relative to the tubular member 120. The first shaft portion 110L of the rotary shaft 110 supporting the left massaging member 116L is turnable relative to the tubular member 120 forming the half-turn clutch 121 within a range of a half turn, while the second shaft portion 110R of the rotary shaft 110 supporting the right massaging member 116R is secured to the tubular member 120 unrotatably relative thereto. Accordingly,

as shown in FIG. 12, when the first shaft portion 110L of the rotary shaft 10 is rotated counterclockwise by the drive element 114 (when viewed from a direction depicted by arrow 127 shown in FIG. 13), the stopper pin 125 comes to abut one radial end face 124a of the semicircular transverse slot 124 and causes the second shaft portion 110R to rotate counterclockwise together with the first shaft portion 110L. When the first shaft portion 110L is rotated clockwise (when viewed as indicated by arrow 127) from the condition in which the stopper pin 25 abuts the radial end face 124a, the stopper pin 125 moves within the transverse slot 124 to abut the other radial end face 124b of the slot 124 and afterward causing the second shaft portion 110R to rotate clockwise together with the first shaft portion 110L.

As the stopper pin 125 moves from the radial end face 124a to the opposite radial end face 124b, the motion of right massaging member 116R mounted on the second shaft portion 110R on the driven side changes relative to the left massaging member 116L. As a result, the massaging members 116R, 116L can assume a non-kneading motion where the two massaging members 116L and 116R move in the same direction parallel with each other as indicated in solid line in FIG. 10, or alternatively a kneading motion where the two members 116L and 116R move in opposite directions as indicated in phantom line in FIG. 10. As shown in FIGS. 13 and 14, respectively, the half-turn clutch 121 forms switching means 126 for selectively switching the motion of the massaging members 116R, 116L into one of the kneading motion, in which the pair of opposite massaging wheels 116L and 116R move opposite one another, and the non-kneading motion, in which they move in the same direction. In other words, by changing the direction of rotation of the rotary shaft 110, the relative motion of the members 116R, 116L is changed thereby changing the type of massage provided by the massaging members 116R, 116L. In alternative embodiments, instead of the half-turn clutch 121, other mechanical electromagnetic or electromechanical switching means or clutches may be incorporated.

In preferred embodiments, the massaging members 116R, 116L are mounted eccentrically, or off-center relative to the rotary shaft 110 such that the lobes 113 of the massaging members 116R, 116L move in a reciprocating fashion relative to the rotary shaft 110. Accordingly, when the rotary shaft 110 is rotably driven from a start position, the lobe 113 of the massaging member 116 exerts pressure on the affected part of the user, which will gradually increase as the rotary shaft 110 rotates through a predetermined angle, 270° example, and then progressively decreases to zero during the remaining 90° of each turn to simulate the massaging actions of the hands of a masseur.

As shown in FIG. 10, the drive unit 114 is driven by a motor 131 (FIG. 6) that includes a gear reduction device 132 for transmitting the driving power of the motor 131 to the first shaft portion 110L of the rotary shaft 110 at a reduced speed. In a preferred embodiment, the gear reduction device 132 is integral with the motor 131. In alternative embodiments, the gear reduction device 132 may be a separate unit from the motor 131.

The gear reduction device 132 includes a gear case 129, a worm wheel 134 and a worm 135. The gear case 129 receives there through the rotary shaft 110 via bearings 130 for rotating the rotary shaft 110. Enclosed within the gear case 129 is the worm wheel 134, which is secured to a portion of the rotary shaft 110. The worm 135 is secured to output shaft 133 of the motor 131 and engaging the worm wheel 134. In this embodiment, the motor 131 can revolve forwards or backwards by way of an electric control circuit not shown. Hence, the forward rotation of the rotary shaft 110 can be switched to the backward rotation, and vice versa. The electric control circuit of the unit 114 is capable of varying the rotary speed of the rotary shaft 110 to at least two levels when the massaging members 116R, 116L are in the non-kneading motion. In one embodiment, the speed varying operation may be effected stepwise. In an alternate embodiment, the speed varying function may be mechanical.

In the counterclockwise non-kneading motion, as illustrated in FIG. 13, the massaging members 116 translate from side-to-side parallel to each other. As the lobes 113 of the members 116R, 116L reciprocate relatively slowly in an alternate fashion, a finger pressure like massage is provided such as to press an affected part of the user heavily from the right and left. To achieve such a finger pressure-like massage, the rotary speed of the rotary shaft 110 is set to about 50 rpm. On the other hand, rotating the rotary shaft 110 at a relatively high speed with the massaging members 116R, 116L in the non-kneading motion causes the lobes 113 of the members 116L, 116R to reciprocate alternately at a higher speed, thereby giving impacts to the affected part of the user, resulting in a tapping massage. To achieve such a tapping massage, the rotary speed of the rotary shaft 110 is set to 150 rpm or higher. Further, the rotary speed of 200 rpm provides the user with a particularly advantageous tapping massage.

In the clockwise kneading motion, as illustrated in FIG. 14, the massaging members 116R, 116L translate from side-to-side, with the lobes 113 of the massaging members 116R, 116L gradually coming closer to each other while reciprocating, and subsequently retracting while going away from each other. In this motion, a kneading massage is provided. The rotary speed of the rotary shaft 110 is preferably set within a range from about 50 to about 60 rpm in the kneading massage.

Referring to FIG. 10, since the first and second shaft portions 110L and 110R are interconnected through the half-turn clutch 121, the second shaft portion 110R can rotate relative to the first shaft portion 110L undesirably due to the pressure imposed on the right massaging member 116R from the affected part of the user. As a result, the position of the massaging member 116R may shift to a position creating a motion (i.e., a kneading or non-kneading motion) that is different from the user selected motion. To prevent such inconveniences, the massaging unit 6 incorporates a first brake system 139 for providing a frictional resistance against rotation of the second shaft portion 110R on the driven side, which is not driven by the drive unit 114. Additionally, a second brake system 140 is used for providing frictional resistance against rotation of the pair of massaging members 116R, 116L relative to the rotary shaft 110.

The first brake system 139 comprises a friction wheel 137 attached to the projecting end of the second shaft portion 110R, and a pressing spring 138 secured to the carriage 56 so that an end portion thereof presses upon the outer periphery of the friction wheel 137. Braking is accomplished by the frictional forces between the frictional wheel 137 and the pressing spring 138. The frictional forces act to retard the rotational momentum of the rotary shaft 110 and bring the shaft to rest.

The second brake system 140 employed in this embodiment comprises a ring spring 155 disposed on opposite sides of each massaging members 116R, 116L. The ring spring 155 is inserted into a clearance between each sandwiching plate 115a, 115b and each massaging members 116R, 116L to provide a friction resistance against the rotation of the members 116R, 116L about the rotary shaft 110. As such, secondary braking is accomplished by pressing the respective slanted faces of the sandwiching plates 115a and 115b upon each massaging members 116R, 116L with an appropriate pressure.

The massaging unit 6 according to this embodiment is capable of selectively performing the kneading massage and other massaging operations by simply switching the rotational direction of the rotary shaft 110. Further, by simply varying the rotary speed of the rotary shaft 110 when the massaging members 116R, 116L are in the non-kneading motion, the massage device can selectively perform the finger pressure-like massage and the tapping massage. Thus, the massaging members 116L, 116R, of a single kind, may perform three different kinds of massaging operations.

For the embodiment shown in FIG. 5, the user, through the use of a controller, can translate the carriage 56 to an appropriate location within a chair back for massaging a specific location of the user's back. This may be accomplished by engaging the drive unit that rotates the

guide rod 90 relative to the cylindrical member 92. Accordingly, the cylindrical member 92 threads along the guide rod 90. The user selects the type of massage desired when the carriage reaches the appropriate location. Depending on the selection, the controller causes the massaging members 116R,116L to rotate in the appropriate direction (i.e., clockwise or counterclockwise) and at the appropriate speed.

The relative mounting of the massage members 116R,116L to the shaft 110 is given herein by way of example. It may be, for example, that the members 116R,116L are mounted such that counterclockwise rotation of the members 116R,116L (when viewed from the direction depicted by arrow 127 as shown in FIG. 10), would cause the two massaging members 116R,116L to move in a parallel fashion, or the members 116R,116L may be mounted such that rotation in a counterclockwise direction (when viewed from the direction depicted by arrow 127 in FIG. 13) would cause the members 116R,116L to orient themselves in a non-parallel relationship such that they are slanted towards each other. Moreover, the type of massages to be given by the massaging members 116R,116L can be further controlled by controlling the degree of the relative eccentricity of the two massaging members 116R,116L relative to the shaft 110.

Referring to FIG. 15, the massaging device 7 of the present invention can be incorporated in a conventional recliner 200. It is preferred that the conventional recliner has a frame 203 on its back 202 to accept the massaging device 7. In a preferred embodiment the frame 203 comprises opposing faces 214 and 216, each face comprising a pair of apertures 212. A pair of fasteners 218 are displaced along the proximal 250 and distal 260 ends of the bracket 32 for engaging the apertures 212 and retaining the massaging device 12 within the back 202 of the recliner 200.

As shown in FIG. 16, each fastener 218 comprises a sliding body 222 and nut plate 230. The body 222 comprises a V-shaped profile 228, for mating the first raceway 52, and a threaded aperture 226, located in a central portion of the body 222. The sliding body 222 preferably comprises aluminum, but may be made of any suitable material. A tubular shank 224 extends from an end of the body 222 for engaging the aperture 212. The nut plate 230 comprises a V-shaped groove 232, for mating the underside of the first raceway 52, and a threaded bore 236, located in a central portion of the plate 230. The nut plate 230 preferably comprises aluminum, but may be made of any suitable material. The fastener 218 is adjustable, as the sliding body 222, and nut plate 230 are coupled by threaded member 242 to translate in unison along the first raceway 52. The threaded member 242 engages aperture 226 and bore 236 within a notch 246 in the first raceway 52, defining the fastener's 218 translation. The fastener 218 is fixed in a

particular position by engaging the threaded member 242 within the aperture 226 and bore 236, causing the profile 228 and groove 232 to contact the first raceway 52.

As shown in FIG. 17, the massaging device 7 is preferably removed from the recliner 200 by loosening the fasteners 218 on the proximal end 250 of the bracket 32. The fasteners 218 will then disengage the pair of apertures 226 on the face 216 of the frame 203. A slit defining a handle 248 is located at the proximal end 250 of the bracket, allowing the user to handle the device 7, tilt it, and remove the unit from the back 202 of the recliner 200. The massaging device 7 can then be transported for service or maintenance as a modular unit. Similarly, after maintenance, the massaging device 7 may be installed into the back 202 of the recliner 200, by engaging the pair of fasteners 218 on the distal end 260 of the bracket 32 into corresponding apertures on the face 214 of the frame 202. Using the handle 248, the pair of fasteners 218 on the proximal end 250 of the bracket 32 are aligned with the pair of apertures 226 on the face 216 of the frame 203. The fasteners 218 are then adjusted to engage the apertures 226 and the threaded members 242 are tightened to hold the fasteners 218 in place.

Referring to FIG. 18, the controller 102 is retained in the back 203 of the recliner 200, along a face 215 of the frame 203, by a retainer bracket 270. The retainer bracket 270 is preferably sheet metal, forming substantially rectangular sidewalls 272, 274 and 275 and fold 278. Sidewalls 272 and 276, each comprise holes 286 aligned with each along a portion of the sidewalls 272 and 276. Ends 282 and 284 of a substantially U-shaped retainer rod 280 are rotably coupled to the holes 286, enabling a central portion 288 of the retainer rod 280 to rotate about the center of the holes 286.

The controller 102 comprises a housing 290 having a flange 292, extending from the base of a front portion of the housing 290, and a pair of clasp 294, coupled along a rear portion of the housing 290.

Referring to FIG. 19, when installed, the flange 292 of the controller 102 engages the fold 278 (not shown) and the retainer rod 280 is rotated, such that the central portion 288 of the retainer rod 280 is fastened within the clasp 294. The controller 102, further, comprises a cutaway 296, allowing the user access to handle the retainer rod 280.

The massaging device of the present invention can also be incorporated in a stand-alone or one-piece back rest as shown in FIGs 20 and 21. A stand-alone or one-piece casing 162 should have longitudinal length substantially corresponding to that of the back of a human. Such a one-piece device may be leaned against a wall W or against the back of a chair 164 for providing a massage. The overall configuration of the casing 162 used in this embodiment is a longitudinally

elongated flat box. This configuration allows for easy storage in narrow spaces such as in a corner of a room or between furniture articles.

5 The massaging members 116L,116R may each be differently varied in configuration so long as the overall configuration thereof is substantially discoid, for example, in the form of an elliptic disc or a polygonal disc. In alternative embodiments, the lobes 113 of the massaging members 116R,116L may be configured in the form of a combination finger and fist. In this embodiment, the boss portion 115 is rotably mounted to the rotary shaft 110, such that the finger
10 configuration may be used, while the fist configuration is positioned out of use. Alternatively, the boss portion 115 may be fixed about the rotary shaft 110, such that the fist configuration may be used, while the finger configuration is positioned out of use. In an additional embodiment, the lobes 113 of the massaging members 116R,116L may be detachable elements in the form of a fist, finger or the like. The members would be fastened to and detachable from the central portion of
15 the massaging members 116R,116L.

Moreover, instead of two massaging members, one or more massaging members may be incorporated in the massaging device. For example, many smaller massaging wheels 157 may be coupled to shafts 150. These shafts 150 are coupled to the massaging unit 6 in parallel to the rotary shaft 110, as shown in FIG. 22.

20 The massaging device of the present invention, incorporating non-rotary massaging members partially discoidal in shape, provides a profile thinner than massaging devices of the prior art. Having non-rotary massaging members are advantageous because only the portion of the member that contacts the affected part of the user require a large radial peripheral. Further, the substantially radial cross-section of the massaging members of the present invention is such that
25 parts of the user (e.g. a users finger or flesh) will not be pinched between the support frame of the massaging unit and the massaging members. Moreover, the use of massaging members comprising polished plastic minimizes frictional contact between the massaging members and the affected chair fabric, and thus reduces wear on the chair fabric.

30 If desired, the massaging unit 6 of the present invention may be translated along a track forming two C-shaped rails. The biasing wheel 72 of the present invention may also be coupled to a massaging unit translated along a track forming two C-shaped rails. Further, the diamond shaped guide wheels 60 and biasing wheel 72 of the present invention may be coupled to a messaging unit comprising a pair of message wheels. A description of such a track and massaging unit are described in PCT International Application No. PCT/JP99/01340 (filed March
35 17, 1999), the disclosure of which is incorporated herein by reference.

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